# Stress Intensification Factors and Flexibility Factors for Pad-Reinforced Branch Connections

TR-110755

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### **REPORT SUMMARY**

This report presents the results of an investigation into the effects of pad reinforcement on the flexibility and fatigue life of fabricated tees. Equations are provided, based on analyses and test data, for determining stress intensification factors and flexibility factors. The expressions presented in this report significantly improve the evaluation of the fatigue life of pad-reinforced branch connections.

#### Background

Fatigue is an important concern in the design and engineering of piping systems. The ASME Section III Class 2 & 3, and B31 piping design codes use factors such as stress intensification factors to account for fatigue effects produced by reversing loads. These factors are known to **often** be conservative. The present version of the ASME Code does not clearly define the flexibility factors for all configurations of branch connections.

#### **Objectives**

- To derive expressions for stress intensification factors (SIF) for pad-reinforced branch connections
- To derive expressions for flexibility factors for accurately modeling the behavior of pad-reinforced branch connections in a piping analysis

#### Approach

A detailed investigation of the behavior of unreinforced fabricated branch connections has been documented in EPRI report TR-110996. Researchers conducted fatigue tests on pad-reinforced branch connections and compared the results to the predicted behavior of the unreinforced branch connections. The differences in results were used to quantify the effects of pad reinforcement.

#### Results

Adjustments were made to the SIF equations in TR-110996. An adjusted thickness term, T\*, was introduced to account for the pad reinforcement. This term accounts for the reduction in SIF due to reinforcement. This new equation is valid for both in-plane and

out-of-plane bending and is valid for all loading conditions on the branch and run pipe. Parameter limitations were established for the results to be applicable to various pad configurations.

Regarding flexibility factors, the T\* term was also applied to previously derived equations in TR-110996 to evaluate the branch pipe. However, it was determined that there does not appear to be sufficient data to justify adjusting the flexibility of the run pipe for the effect of the pad-reinforced branch connection.

#### **EPRI** Perspective

Design for fatigue is a major concern for any power or process facility. Accurate methods of engineering for fatigue are important for cost-effective design, for root cause failures, and for evaluating remaining fatigue life of plant designs. The work being done under EPRI's SIF optimization program continues to establish the technical justification to allow for reductions in current Code stress intensification factors. The results of this program can provide a basis to reduce the scope of ongoing pressure boundary component testing and inspection programs in operating nuclear power plants. Examples include reductions in the inspection scope of postulated high- and moderate-energy line break locations and reduction of snubber testing.

#### TR-110755

#### **Interest Categories**

Piping, reactor vessel, and internals

### ABSTRACT

This report was prepared under the auspices of the EPRI (Electric Power Research Institute) project on Stress Intensification Factor Optimization. The fatigue life of branch connections is a major consideration in the design and evaluation of piping systems. This report presents the results of an investigation of the effects of pad reinforcement on the flexibility and fatigue life of fabricated tees subject to various moments. The report reviews existing test data and develops expressions for estimating SIFs and flexibility factors. The expressions presented in this report significantly improve the evaluation of the fatigue life of pad-reinforced branch connections.

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# **1** INTRODUCTION

The configuration of a pad-reinforced fabricated branch connection is shown in Figures 1-1 and 1-2. A pad-reinforced branch connection can be visualized as an unreinforced fabricated branch connection with a collar added at the base of the branch segment. A detailed investigation of the behavior of unreinforced fabricated branch connections has been conducted under this EPRI-sponsored research effort [1]. This study presents the results of fatigue tests conducted on pad-reinforced branch connections, and compares the results to the predicted behavior of an unreinforced fabricated branch connection. The differences in results are used to quantify the effects of pad reinforcement.



Figure 1-1 Pad-Reinforced Fabricated Branch Connection

#### 1.1 Nomenclature

- $A_{0}$  = constant used in expression for SIFs
- $B_0 = constant$  used in expression for flexibility
- D = mean diameter of run pipe, inches
- d = mean diameter of the branch pipe, inches

#### Introduction

- $D_{o}$  = outside diameter of the run pipe, inches
- $D_{op}$  = diameter of pad, inches
- $d_{\circ}$  = outside diameter of the branch pipe, inches
- F = force applied to test specimens, pounds
- K = overall stiffness of test configurations, pounds/inch
- $n_1$ ,  $n_2$ , and  $n_3$  = constants
- T = wall thickness of the run pipe, inches
- $T^*$  = adjusted wall thickness of the run pipe, inches
- t = wall thickness of the branch pipe, inches
- t<sub>r</sub> = thickness of reinforcement pad, inches
- t<sub>e</sub> = effective fitting thickness, inches
- R = mean radius of the run pipe, R =  $(D_0 T)/2$ , inches
- r = mean radius of the branch pipe, r =  $(d_0-t)/2$ , inches
- R/T = characteristic of the run pipe
- r/t = characteristic of the branch pipe
- r/R = characteristic of the connection
- t/T = characteristic of the connection
- $Z = approximate section modulus of the run pipe, in<sup>3</sup>, =\pi R<sup>2</sup>T$
- $z = approximate section modulus of the branch pipe, in<sup>3</sup>, =<math>\pi r^{2}t$
- M<sub>ir</sub> = in-plane bending moments on the run, in-lb.
- M<sub>or</sub> = out-of-plane bending moments on the run, in-lb.
- $M_{\rm tr}$  = torsion moments on the run, in-lb.

#### Introduction

 $M_{ib}$  = in-plane bending moment on the branch, in-lb.

 $M_{ob}$  = out-of-plane bending moment on the branch, in-lb.

 $M_{tb}$  = torsion moment on the branch, in-lb.

i<sub>ib</sub> = Stress Intensification Factor for in-plane bending moments on the branch pipe

i<sub>ob</sub> = Stress Intensification Factor for out-of-plane bending moments on the branch pipe

 $i_{tb}$  = Stress Intensification Factor for torsion moments on the branch pipe

i<sub>ir</sub> = Stress Intensification Factor for through-run in-plane bending moments on the run pipe

 $i_{or}$  = Stress Intensification Factor for through-run, out-of-plane bending moments on the run pipe

itr = Stress Intensification Factor for through-run torsion moments on the run pipe

 $k_{ib}$  = Flexibility Factor for in-plane bending moments on the branch pipe

 $k_{ob}$  = Flexibility Factor for out-of-plane bending moments on the branch pipe

 $k_{tb}$  = Flexibility Factor for torsion moments on the branch pipe

kir = Flexibility Factor for through-run in-plane bending moments on the run pipe

 $k_{or}$  = Flexibility Factor for through-run, out-of-plane bending moments on the run pipe

 $k_{tr}$  = Flexibility Factor for through-run, torsion moments on the run pipe

#### Introduction



Figure 1-2 Cross-section of Pad-Reinforced Fabricated Tee

#### 1.2 Current Code Factors

The current Code expression [2,3] for the Stress Intensification Factor for a padreinforced branch connection is:

$$i = 0.9/h^{2/3}$$
 (eq. 1-1)

where:

$$h = (T + t_r/2)^{3/2} / (Rt_r^{3/2})$$
(eq. 1-2)

Equation 1-1 values are applied to the resultant moment on each of the three sides of the tee connection.

These expressions are based on Markl's work [4]. Markl began with the premise that the i-factor was of the form:

 $i=0.9/h^{2/3} \ge 1$ 

where

 $h=c(t_{e}/R)$  and  $c=(t_{e}/T)^{1.5}$ 

Markl called h, the *effective flexibility factor*, and said that c *takes account of the increased section modulus*. Both of these factors are a function of the *effective fitting thickness* at the juncture,  $t_e$ , which Markl saw to be the average of the thickness of the run pipe and the thickness of the branch pipe ( $t_e = (T+t)/2$ ). For a pad thickness,  $t_r$ , and for a connection

with t=T, then  $t_e = (T+t_r+T)/2 = T+t_r/2$ . Thus, Markl determined  $t_e$  to be the pipe wall thickness increased by one half the excess thickness, t<sub>e</sub>, provided in either the run or branch, by use of thicker piping, or pad, or saddle. Note that Markl's statement is for t=T.

In 1965, ANSI Code Case 53 [5] introduced rules for evaluation of reduced branch outlet tee connections. These rules use  $z=\pi r^2 t_s$  where  $t_s$  is the lesser of T or i\*t, instead of  $z=\pi r^2 t_s$  for evaluation of the branch side resultant moment. The Code Case is now incorporated in the sections of the Code on determination of moments and section modulus of tee connections. The effect of the Code Case is to change Equation 1-1 to the following two equations:

$$\begin{split} i_{run} &= 0.9 R^{2/3} \left[ T/(T+t_r/2)^{5/3} \right] \geq 1.0 \end{split} \tag{eq. 1-3} \\ i_{branch} &= 0.9 R^{2/3} \left[ T/(T+t_r/2)^{5/3} \right] (t/T) \geq 1.0 \end{aligned} \tag{eq. 1-4}$$

Equation 1-3, is used with  $Z=\pi R^2 T$  to evaluate the resultant moments on the run sides of the tee connection, and Equation 1-4 is used with  $z=\pi r^2 t$  to evaluate the resultant moment on the branch side of the tee connection. Equations 1-3 and 1-4 are applied over the entire range of  $0 < r/R \le 1.0$ . The current Code expression for the i-factor is independent of any reinforcement pad dimension other than the thickness of the pad.

# **2** EXPERIMENTAL RESULTS

#### 2.1 Markl Tests

Markl tested 4x4 pad-reinforced branch connections with the following results. All branch connections had an outside diameter,  $D_0 = d_0 = 4.5$  inches and wall thicknesses were either T= t=0.237 inches, or T=t=0.12 inches. The reinforcement pads were either elliptical or circular in shape.

Test	Pad thickness, t <sub>r</sub> Pad Outside Diameter		Load Direction	$\mathbf{i}_{ ext{test}}$
1	0.237 inches	7 inches	in-plane branch	1.78
2	0.237 inches	7 inches	out-of-plane branch	1.83
3	0.120 inches	8 inches	in-plane branch	2.21
4	0.120 inches	8 inches	out-of-plane branch	2.43

Markl also cited a fatigue test by Blair [6] conducted on a  $D_0 = d_0 = 6.5$  inches, t=T=0.26 inches, pad-reinforced tee with the following result:

Test	Pad thickness, $t_r$	d thickness, t <sub>r</sub> Pad Outside Diameter		$\mathbf{i}_{test}$
5	0.312 inches	10.75 inches	in-plane branch	2.58

Markl computed i-factors for these specimens, using Equation 1-1, of

Test	$\mathbf{i}_{ ext{test}}$	Eq. 1-1 i-factor	% Difference	
1	1.78	1.98	11%	
2	1.83	1.98	8%	
3	2.21	3.17	44%	
4	2.43	3.17	31%	
5	2.58	2.16	-16%	

#### 2.2 EPRI Tests

#### 2.2.1 Objectives

The most obvious void in Markl's information is the lack of fatigue tests on reduced branch outlet tees. To fill this void, fatigue tests on reduced branch outlet fabricated branch connections were conducted under this research effort.

#### 2.2.2 Specimens

In-plane bending of the branch fatigue tests were conducted on four 8x4 inch NPS fabricated tees reinforced with 0.25 inch thick circular pads. The specimens were manufactured by Wilson Welding Service, Inc., of Decatur, GA. The reinforcement pad outside diameter was varied among the four specimens to examine the effect of this dimension. The specimen configurations are illustrated in Figure 2-1.



Figure 2-1 Test Configuration

#### 2.2.3 Testing

The testing was performed at the Ohio State University. See reference [7] for a description of the test equipment and methodology.

The test distance from the load point to the surface of the pipe (~46 inches) varies slightly for each test specimen. The measured distance that is dependent on the installation is included in the test data.

The test data, results, and other information are provided in Appendix C. The tests were displacement-controlled cantilever bending tests. The tests followed the standard approach corresponding to Markl type tests [4]. Each specimen was first tested to determine the load deflection curve for that particular specimen. The load deflection curve was used to determine the stiffness of each specimen, and the load applied to the specimen by a given amount of displacement. The load deflection curves were determined for loading in both positive and negative loading directions (down and up). Each specimen was then fatigue tested by cycling the deflection in both directions of loading by a controlled amount. The cycles to failure were counted to determine the fatigue life. Failure was detected when throughwall cracks formed at the toe of the filet weld in the longitudinal plane and water leaked through the crack.

#### 2.2.4 Fatigue Test Results

7.5

Appendix C of this report. The langue lest results for each specifient were as follows.								
Specimen	Pad OD	Nominal Stress (M/z)	Cycles to Failure	$\mathbf{i}_{test}$				
F	6	26 773	1 260	2 20				
F	7	23,997	1,901	2.26				
G	9.5	28,752	1,366	2.01				

28.133

Testing was conducted at Ohio State University. The test report is provided in Appendix C of this report. The fatigue test results for each specimen were as follows:

The value of i is calculated from i = 245,000 N<sup>-0.2</sup>/S, where N = cycles to failure, and S = M/z = FL/z. The value of z in the calculation of the nominal stress is given by the expression  $z=\pi r^2 t$  for consistency with the i-factors determined for unreinforced branch connections.

1.075

#### 2.2.5 Stiffness Results

Н

As discussed above, the stiffness of each of the four EPRI test specimens was determined experimentally and is given below. The width of the pad,  $D_{op}$ -d<sub>o</sub>, divided by the "attenuation length"  $2(D_o t_p)^{0.5}$  is also given for comparison.

2.16

#### **Experimental Results**

Specimen	Stiffness (lbs∕in)	D <sub>op</sub> (in)	$(D_{op}-d_{o})/(2(D_{o}t_{r})^{0.5})$
E	1963	6.0	0.51
F	2018	7.0	0.85
G	2109	9.5	1.70
Н	2092	7.5	1.02

The average stiffness is 2046 lbs/in. The range from this average value is +4 to -3%. This test series suggests that if the pad width divided by the attenuation length  $2(D_o t_r)^{0.5}$  is greater than approximately 0.5, further increases in pad outside diameter,  $D_{op}$ , have no effect on i-factors or k- factors.

# **3** STRESS INTENSIFICATION FACTORS

#### 3.1 Discussion

Pad-reinforced branch connections are similar to unreinforced branch connections. It is expected that, with an adjustment to account for the effects of the pad, the equations for the SIFs should be very similar. Consequently, the approach used in this study is to compare the data for the pad-reinforced branch connections to the results from the study on unreinforced branch connections and determine if an adjustment considering the reinforcement would result in a methodology that would be applicable to the padreinforced connections.

#### 3.2 Unreinforced Fabricated Branch Connections

Reference [1] provides a detailed study of the SIFs for unreinforced fabricated branch connections. The equations for SIFs are developed based upon test data and extensive finite element analysis. The conclusions of the report contain the equations for the SIFs for different loading conditions, discussion on the combination of moments, and the applicability of the results. This information is summarized below:

For individual loadings, for example, torsion, in-plane and out-of-plane bending, the stress intensification factors for the branch and run pipe ends are given by:

$i_{xy} = A_o (R/T)^{n1} (r/t)^{n2} (r/R)^{n3} \ge 1.0$										
	Part	SIF	A	n <sub>1</sub>	n <sub>2</sub>	n <sub>3</sub>				
In-plane bending	Branch Run	i <sub>ib</sub> i <sub>ir</sub>	0.515 0.985	1.05 -0.137	-0.387 0.482	0.49 0.241				
Out-of- plane bending	Branch Run	${f i}_{_{ m ob}}$ ${f i}_{_{ m or}}$	Note 1 0.605	1.40 -0.237	-0.558 0.528	0.406 1.42				
Torsion	Branch Run	i <sub>tb</sub> i <sub>tr</sub>	0.850 0.864	1.00 -0.0473	-0.50 0.543	2.10 0.609				

Note 1: Replace  $A_0$  with 1.28[1.28(r/R)-(r/R)<sup>4</sup>]

(eq. 3-1)

#### Stress Intensification Factors

For combination of stresses due to different moments, the various code committees are considering changes at this time. Until these changes are finalized, the evaluation of the stresses must be based on the maximum i-factor for the branch and the maximum i-factor of the run. If Section III is the Code of record (for Class 2 or 3 piping), Equation 3-2 should be used for evaluating the branch connections where the branch and each end of the run pipe are evaluated.

$$S = \frac{i(M_i^2 + M_o^2 + M_t^2)^{1/2}}{Z_i}$$
 (eq. 3-2)

The values of the moments and section modulus are appropriate for the location. The value of  $Z_i$  for the branch is given by the equation :

$$z = \pi r^2 t \tag{eq. 3-3}$$

For the run pipe, Z<sub>i</sub> is:

$$Z = \pi R^2 T$$
 (eq. 3-4)

For the branch, the value of i is the maximum of  $i_{ib}$ ,  $i_{ob}$ , or  $i_{tb}$ . For the run, the value of i is the maximum of  $i_{ir}$ ,  $i_{or}$ , or  $i_{tr}$ .

If B31.1 is the Code of record, a less conservative approach is permitted. This is discussed in reference [1].

The equations specified above in Equation 3-1 are applicable to the following range of parameters [1]:

$3.75 \leq R/T \leq 49.5$	(eq. 3-5)
$3.75 \leq r/t \leq 99$	(eq. 3-6)
$0.125 \leq r/R \leq 1.0$	(eq. 3-7)

#### 3.3 Extension to Pad-Reinforced Branch Connections

Table 3-1 lists the fatigue test results for pad-reinforced branch connections. The tests labeled 1, 2, 3, 4, and 5 were conducted or cited by Markl. The tests labeled E, F, G, and H are the EPRI-sponsored tests. The tests labeled K1, K2, K3, and K4 are from Khan [8]. The type of loading (in-plane or out-of-plane loading) is indicated. All loads were applied to the branch. The column titled " $i_{test}$ " lists the experimentally determined i-factors.

Table 3-2 contains a comparison of the test i-factors to those derived from the present Code (labeled  $i_{Code}$ ) and to the expressions for the unreinforced fabricated branch

connections given in Equation 3-1 above (labeled  $i_{uft}$ ). The columns marked "% Diff" contain the percentage differences between the test data and either  $i_{Code}$  or  $i_{uft}$ . Note that the form of the equation for  $i_{uft}$  used in Table 3-2 corresponds to the type of loading, in other words, loading on the branch, either in-plane or out-of-plane.

The average, maximum, and minimum percentage differences are also listed in Table 3-2.

#### Stress Intensification Factors

								Pad OD								
Test	Load	Do	Т	d <sub>o</sub>	t	t,	t <sub>e</sub>	D <sub>op</sub>	R	r	R/T	r/t	r/R	t/T	t <sub>r</sub> /T	i <sub>test</sub>
	Direction	(in.)	(in.)	(in.)	(in.)	(in.)	(in.)	(in.)	(in.)	(in.)						
1	in-plane	4.500	0.237	4.500	0.237	0.237	0.356	7.00	2.132	2.132	8.99	8.99	1.000	1.000	1.00	1.78
2	out-of-plane	4.500	0.237	4.500	0.237	0.237	0.356	7.00	2.132	2.132	8.99	8.99	1.000	1.000	1.00	1.83
3	in-plane	4.500	0.120	4.500	0.120	0.120	0.180	8.00	2.190	2.190	18.25	18.25	1.000	1.000	1.00	2.21
4	out-of-plane	4.500	0.120	4.500	0.120	0.120	0.180	8.00	2.190	2.190	18.25	18.25	1.000	1.000	1.00	2.43
5	in-plane	6.500	0.260	6.500	0.260	0.312	0.416	10.75	3.120	3.120	12.00	12.00	1.000	1.000	1.20	2.58
E	in-plane	8.625	0.250	4.500	0.237	0.250	0.375	6.00	4.188	2.132	16.75	8.99	0.509	0.948	1.00	2.20
F	in-plane	8.625	0.250	4.500	0.237	0.250	0.375	7.00	4.188	2.132	16.75	8.99	0.509	0.948	1.00	2.26
G	in-plane	8.625	0.250	4.500	0.237	0.250	0.375	9.50	4.188	2.132	16.75	8.99	0.509	0.948	1.00	2.01
Н	in-plane	8.625	0.250	4.500	0.237	0.250	0.375	7.50	4.188	2.132	16.75	8.99	0.509	0.948	1.00	2.16
K1	out-of-plane	8.625	0.322	3.500	0.216	0.322	0.483	6.00	4.152	1.642	12.89	7.60	0.396	0.671	1.00	1.95
K2	out-of-plane	8.625	0.322	6.625	0.280	0.322	0.483	12.00	4.152	3.173	12.89	11.33	0.764	0.870	1.00	2.78
K3	out-of-plane	12.75	0.375	6.625	0.280	0.375	0.563	12.00	6.188	3.173	16.50	11.33	0.513	0.747	1.00	3.47
K4	out-of-plane	12.75	0.375	8.625	0.322	0.375	0.563	15.00	6.188	4.152	16.50	12.89	0.671	0.859	1.00	3.91
Notes:																
1. This	table is prod	uced on	a spread	sheet us	ing EXCE	L. The	number	of signific	ant figure	es is grea	ater than	n indicat	ted.			
2. All I	oads are on th	ne brand	ch.													

# Table 3-1Test Data for Pad-Reinforced Fabricated Tees

# Table 3-2Comparison to Unreinforced Branch Connections and Code Expressions

Test	Load	R/T	r/t	r/R	i <sub>test</sub>	i <sub>uft</sub>	%Diff	i <sub>Code</sub>	%Diff			
	Direction											
1	in-plane	8.99	8.99	1.000	1.78	2.20	19.1	1.98	11.2			
2	out-of-plane	8.99	8.99	1.000	1.83	2.28	19.7	1.98	8.2			
3	in-plane	18.25	18.25	1.000	2.21	3.51	37.1	3.17	43.6			
4	out-of-plane	18.25	18.25	1.000	2.43	4.13	41.2	3.17	30.6			
5	in-plane	12.00	12.00	1.000	2.58	2.66	3.1	2.16	-16.5			
E	in-plane	16.75	8.99	0.509	2.20	3.04	27.7	2.84	29.5			
F	in-plane	16.75	8.99	0.509	2.26	3.04	25.7	2.84	26.0			
G	in-plane	16.75	8.99	0.509	2.01	3.04	33.8	2.84	41.3			
Н	in-plane	16.75	8.99	0.509	2.16	3.04	29.0	2.84	31.8			
K1	out-of-plane	12.89	7.60	0.396	1.95	2.18	10.4	1.69	-13.4			
K2	out-of-plane	12.89	11.33	0.764	2.78	2.57	-8.1	2.19	-21.3			
K3	out-of-plane	16.50	11.33	0.513	3.47	2.74	-26.6	2.22	-36.1			
K4	out-of-plane	16.50	12.89	0.671	3.91	2.97	-31.5	2.55	-34.8			
						AVE =	13.9	AVE =	22.9			
						MAX=	41.2	MAX=	43.6			
						MIN=	-31.5	MIN=	-16.5			
Notes:												
1. This	table is produ	uced or	a spreads	sheet us	ing EXCE	EL.						
The nu	mber of signifi											
2. i <sub>uft</sub> is	2. i <sub>uft</sub> is determined by Equation 3-1 in this report.											

#### 3.4 Adjusted Thickness

It appears reasonable to assume that the pad thickness effectively increases the thickness of the run pipe and hence should act to reduce the SIF. Markl [4] considered this by using a larger "effective thickness" in determining SIFs for pad-reinforced branch connections. Following this approach, the use of an "effective thickness" will be considered.

The expressions derived for unreinforced branch connections are in terms of the parameters: (R/T), (r/t), and (r/R). The effectiveness of the reinforcement of the run pipe is perhaps best reflected in the (R/T) term. Thus (R/T) term will be replaced by  $(R/T^*)$  where

$$T^* = T + Ct_r$$
 (eq. 3-8)

"C" is a constant to be determined.

The value of "C" was selected based on an evaluation of the percentage difference (% Diff) between the modified values of i and the test data. Table 3-3 shows the effects of adjusting the thickness on the SIFs (labeled i-pad in Table 3-3). It was determined that for C = 1/2, the average percentage difference was 0.9%. The range was from -37% to +42%.

Thus, Equation 3-1 should be replaced by:

$$i_{xy} = A_o (R/T^*)^{n_1} (r/t)^{n_2} (r/R)^{n_3} \ge 1.0$$
 (eq. 3-9)

where

$$T^* = T + t_r/2$$
 (eq. 3-10)

While the test data is primarily for in-plane bending of the branch, it is reasonable to assume that the use of  $T^*$  is valid for all loading conditions on the branch since the SIFs are based on the same theoretical approach. The same assumptions should also be applicable to the SIFs for the loads on the run pipe.

#### Stress Intensification Factors

Test	Load	Т	T*=T+C tr	R/T	R/T*	r/t	r/R	i-pad	i <sub>test</sub>	%Diff	i-pad/i-test	
1	in-plane	0.237	0.356	8.99	6.00	8.99	1.000	1.44	1.78	-19.3	0.81	
2	out-of-plane	0.237	0.356	8.99	6.00	8.99	1.000	1.29	1.83	-29.4	0.71	
3	in-plane	0.120	0.180	18.25	12.17	18.25	1.000	2.29	2.21	3.8	1.04	
4	out-of-plane	0.120	0.180	18.25	12.17	18.25	1.000	2.34	2.43	-3.6	0.96	
5	in-plane	0.260	0.416	12.00	7.50	12.00	1.000	1.62	2.58	-37.0	0.63	
E	in-plane	0.250	0.375	16.75	11.17	8.99	0.509	1.98	2.20	-9.7	0.90	
F	in-plane	0.250	0.375	16.75	11.17	8.99	0.509	1.98	2.26	-12.1	0.88	
G	in-plane	0.250	0.375	16.75	11.17	8.99	0.509	1.98	2.01	-1.4	0.99	
Н	in-plane	0.250	0.375	16.75	11.17	8.99	0.509	1.98	2.16	-8.0	0.92	
K1	out-of-plane	0.322	0.483	12.89	8.60	7.60	0.396	2.77	1.95	42.2	1.42	
K2	out-of-plane	0.322	0.483	12.89	8.60	11.33	0.764	3.83	2.78	37.9	1.38	
K3	out-of-plane	0.375	0.563	16.50	11.00	11.33	0.513	4.24	3.47	22.3	1.22	
K4	out-of-plane	0.375	0.563	16.50	11.00	12.89	0.671	4.92	3.91	26.1	1.26	
									AVE =	0.9		
									MAX=	42.2		
									MIN=	-37.0		
Notes:												
1. This	s table is prod	uced or	n a spreads	heet us	ing EXCE	EL.						
The nu	mber of signifi	cant fig	ures is gre	ater thar	n indicate	ed.						
2. The	value of "i-pao	d" is ca	lculated fro	m Equa	tion 3-1 เ	using the	adjuste	d thickne	ess T*.			

#### Table 3-3 Effects of Adjusted Thickness

#### 3.5 Applicability

In addition to the limits of applicability of the expressions for the SIFs for unreinforced branch connections, (Equations 3-5 to 3-7), there are limits for pad configurations for which the use of the adjusted thickness are valid. Based upon the test configurations, it is suggested that these results are applicable when:

0 ≤	$t_r/T \leq 1.25$	(eq. 3-11)
0 ≤	$D_{op}/d_o \leq 2.0$	(eq. 3-12)

where  $D_{op}$  is the outside diameter of the pad.

# **4** FLEXIBILITY OF PAD-REINFORCED BRANCH CONNECTIONS

#### 4.1 Discussion

In piping design, the flexibility of the piping components can be as significant as the SIFs. Reference [1] discusses the impact of including the flexibility of unreinforced branch connections in an analysis and shows that the loads and stresses can be significantly reduced. The impact of the flexibility of pad-reinforced branch connections can be as significant.

The present Code does not clearly define flexibility factors for all configurations of branch connections. For ASME Class 1, the Code does provide some direction, however, it is not complete. Configurations with r/R > 0.5 are not covered. For Class 2 piping, the flexibility factor is defined to be 1.0 which, as discussed in [1], is misleading at best.

Reference [1] provides guidance (equations) for the determination of flexibility factors for branch connections. As with the SIFs, these equations should be valid for padreinforced branch connections with an adjustment to consider the effects of the pad.

#### 4.2 Unreinforced Fabricated Branch Connections

Reference [1] provides the results of a detailed study of the flexibility factors for unreinforced fabricated branch connections. The equations are based on extensive finite element analyses. The conclusions of the report contain the equations for the flexibility factors for the various loading conditions and the applicability of the results. This information is summarized below:

The flexibility modeling of the branch connection should be based on Figure 4-1 for branch loading. Point springs are assumed to exist to represent the flexibility of the connection. For branch loading, it is assumed there is a rigid link from the centerline of the run pipe to its outer surface. At this point, it is assumed that a point spring exists. The flexibility factors of the point springs are given by:

Flexibility of Pad-Reinforced Branch Connections

	$\mathbf{k}_{xy} = \mathbf{B}_{o}$ (E	$(d / T)^{n1}$	$/D)^{n^{2}}(d/t)^{n^{3}}$			(eq.	4-1)
	Part	k	B <sub>o</sub>	$n_1$	$n_2$	n <sub>3</sub>	
In-plane bending	Branch Run	k <sub>ib</sub> k <sub>ir</sub>	0.488 0.995	1.279 0.675	0.391 3.78	-0.602 -0.250	
Out-of- plane bending	Branch Run	$f k_{_{ob}} \ f k_{_{or}}$	Note 1 0.0771	1.72 -0.159	0.5057 4.096	-0.717 1.305	
Torsion	Branch Run	$f k_{_{ m tb}} \ f k_{_{ m tr}}$	2.43 0.813	0.751 0.982	2.11 4.328	-0.553 -0.349	

Note 1: Replace  $B_0$  with 0.828 (3.0 (d/D)-3.75(d/D)<sup>2</sup>+(d/D)<sup>3</sup>)

Note 2: For d/D<0.5, values of  $k_{ir}$ ,  $k_{or}$ , and  $k_{tr}$  are small and should be set to zero. The equations for  $k_{ir}$ ,  $k_{or}$ , and  $k_{tr}$  are valid only for d/D  $\ge$  0.5.

The equations specified above are applicable to the following range of parameters [1]:

$3.75 \leq R/T \leq 49.5$	(eq. 3-5)
$3.75 \leq r/t \leq 99$	(eq. 3-6)
$0.125 \le r/R \le 1.0$	(eq. 3-7)



Figure 4-1 Beam Model for Branch Loading

#### 4.3 Comparison to Test Data

In a limited manner, the test data from the SIF tests performed for this study can be used to evaluate the applicability of these equations (at least for in-plane loading of the branch connection). It should be noted that these tests were not specifically designed for evaluation of the flexibility factors. For this evaluation, it is assumed that the branch will act as a continuous straight section of pipe. In reality, as indicated in Figure 2-1, the section consists of a section of pipe, pipe flanges, and a "loading point" section, which is made from a flat plate.

The model depicted in Figure 4-1 will be used as the basis for evaluating the test results and determining the flexibility of the branch connection. In this case, the bottom of the run pipe (point 1) is fixed and the top (point 6) is free. The in-plane load is applied on the branch at point 5. As discussed earlier, the "stiffness", K, was determined experimentally from the load deflection data to be, on the average, K = 2046 lbs/in. The range was very small.

The rotation at point 2, with respect to the fixed end (point 1) is given by:

 $\phi_{2-1} = F L_1 L_2 / (EI_r)$ 

(eq. 4-2)

Flexibility of Pad-Reinforced Branch Connections

where F is the applied force.

The deflection at point 5 due to this rotation is:

$$\delta_{a} = \phi_{2-1} L_{1} = F L_{1}^{2} L_{2} / (EI_{r})$$
(eq. 4-3)

The rotation in the point spring is:

$$\phi_{3.4} = k_{ib} M_4 d_o / EI_b = k_{ib} F L_1 d_o / EI_b$$
(eq. 4-4)

where  $M_4$  is the moment at point 4,

The deflection at point 5 due to this rotation is:

$$\delta_{\rm b} = \phi_{\rm 3-4} L_1 = k_{\rm ib} F L_1^2 d_{\rm o} / EI_{\rm b}$$
(eq. 4-5)

The deflection at point 5 due to the bending of the branch is:

$$\delta_{c} = FL_{1}^{3}/(3EI_{b})$$

The total deflection is  $\delta = \delta_a + \delta_b + \delta_c$ , or:

$$\delta = F L_1^2 L_2 / (EI_r) + k_{ib} F L_1^2 d_o / EI_b + FL_1^3 / (3EI_b)$$
(eq. 4-6)

The deflection is also given by:

$$\delta = F/K \tag{eq. 4-7}$$

Substituting Equation 4-7 into Equation 4-6 and solving for  $k_{ib}$  yields:

$$\mathbf{k}_{ib} = [1/K - L_1^2 L_2 / (EI_r) - L_1^3 / (3EI_b)] EI_b / (L_1^2 d_o)$$
(eq. 4-8)

Using K = 2046 lbs./in. (the average of the test data),  $D_0 = 8.625$  in.,  $d_0 = 4.5$  in.,  $L_1 = 46$  in.,  $L_2 = 63$  in. E = 30 E6 psi,  $I_r = 57.7$  in<sup>3</sup>, and  $I_b = 7.23$  in.<sup>3</sup> yields  $k_{ib} = 5.97$ . As discussed

 $L_2 = 63$  in. E = 30 E6 psi,  $I_r = 57.7$  in<sup>3</sup>, and  $I_b = 7.23$  in.<sup>3</sup> yields  $k_{ib} = 5.97$ . As discussed earlier, because test configuration included flanges, etc., this is considered to be a very rough estimate of the flexibility of the test configuration.

Reference [9] lists experimental flexibility data from several sources. Khan [10] also provides experimental flexibility data. This information is summarized in Table 4-1 along with the average data for the tests performed for this study as described above.

When investigating SIFs it was determined that more accurate values could be obtained by using adjusted values of T, in other words,  $T^* = T + t_r/2$  (Equation 3-10) in the

equations for SIFs. Table 4-1 lists the values for  $k_{_{1b}}$  and  $k_{_{ob}}$  calculated using this adjusted value for  $T^* = T + t_r/2$  in Equation 4-1. A comparison to the test data is also provided.

For out-of-plane bending using Equation 4-1 with the adjusted value of T yields results that are reasonably close with an average difference of 10.2%. For in-plane bending, the results are different. The differences on a percentage basis are much greater, however, it should be noted that the magnitudes of the flexibilities are much smaller. The average value of  $k_{_{bb}}$  is 3.8 versus 16.9 for  $k_{_{ob}}$ . It is more difficult to experimentally determine flexibility factors that are small than it is to determine larger values. Also, the potential impact of this order of magnitude difference would not significantly affect the results.

Based upon the overall results, it is deemed reasonable to use Equation 4-1 with the adjusted value of  $T^* = T + t_r/2$  for  $k_{_{ib}}$  and  $k_{_{ob}}$ . In addition, since the theoretical basis is the same, it is reasonable to extend the same approach for the torsional stiffness of the branch,  $k_{_{tb}}$ .

However, there does not appear to be sufficient data to justify adjusting flexibility of the run pipe for the effect of the pad-reinforced branch connection. Consequently, until more data is available, it is suggested that no additional flexibility be added to the run pipe for pad-reinforced branch connections.

	Test	Do	Т	d <sub>o</sub>	t	tr	T*=T+t <sub>r</sub> /2	D/T*	d/D	d/t	k <sub>ob</sub>	k <sub>test</sub>	% Diff	k <sub>ib</sub>	k <sub>test</sub>	% Diff
		(in.)	(in.)	(in.)	(in.)	(in.)	(in.)									
1.	EFGH (Ave)	8.625	0.250	4.50	0.237	0.250	0.375	22.33	0.509	17.99	10.65	N/A	N/A	3.50	5.97	-41.4
2.	24 x 12	24.00	0.312	12.75	0.250	0.375	0.500	47.42	0.528	50.00	18.98	18.0	5.5	5.02	5.60	-10.4
3.	24 x 4	24.00	0.312	4.50	0.237	0.375	0.500	47.42	0.180	17.99	14.19	20.0	-29.1	6.10	4.70	29.8
4.	24 x 8	24.00	0.312	8.625	0.250	0.375	0.500	47.42	0.354	33.50	19.16	28.0	-31.6	5.46	3.60	51.7
5.	16 x 6	16.00	0.500	6.625	0.280	0.500	0.750	20.67	0.409	22.66	6.88	8.40	-18.1	2.53	1.50	68.7
6.	48 x 6	49.25	0.625	6.625	0.280	0.625	0.938	51.87	0.130	22.66	9.27	10.0	-7.3	5.25	1.60	228.1
7.	K1	8.625	0.322	3.500	0.216	0.322	0.483	17.19	0.396	15.20	6.49	3.9	66.4	2.51	N/A	N/A
8.	K2	8.625	0.322	6.625	0.280	0.322	0.483	17.19	0.764	22.66	5.64	4.6	22.6	2.55	N/A	N/A
9.	K3	12.75	0.375	6.625	0.280	0.375	0.563	22.00	0.513	22.66	8.82	5.8	52.1	2.99	N/A	N/A
10	. K4	12.75	0.375	8.625	0.322	0.375	0.563	22.00	0.671	25.79	8.40	6.4	31.3	3.08	N/A	N/A
											AVER	AGE =	10.2	ŀ	AVE =	54.4
											MAX	MUM=	66.4	ſ	MAX =	228.1
											MINI	/UM =	-31.6		MIN =	-41.4
Nc	otes:															
1.	This table is	produce	ed on a	spread	sheet u	sing E>	CEL. The	number	of sign	ificant fi	gures i	s greate	er than	indicate	ed.	
2.	Test "EFGH	(Ave)" r	efers to	the ave	erage re	sults fr	om the four	r tests p	erforme	d as a j	part of t	his stu	dy.			
3.	Tests 2 throu	ugh 6 ar	e from F	Referen	ce 9, N	UREG/	CR-4785.									
4.	Tests 7 throu	ugh 10 a	are from	Khan (	Referer	nce 10).										

# Table 4-1 Test Data for Pad-Reinforced Branch Connections

# 5 CONCLUSIONS

For reinforced branch connections, the conclusions derived from the tests and analysis described in this report are listed below.

#### 5.1 Stress Intensification Factors

For individual loadings, for example, torsion, in-plane and out-of-plane bending, the stress intensification factors for the branch and run pipe ends are given by:

$$i_{xy} = A_o (R/T^*)^{n1} (r/t)^{n2} (r/R)^{n3} \ge 1.0$$

	Part	SIF	A	n <sub>1</sub>	n <sub>2</sub>	n <sub>3</sub>
In-plane	Branch	i <sub>ib</sub>	0.515	1.05	-0.387	0.49
bending	Run	i <sub>ir</sub>	0.985	-0.137	0.482	0.241
Out-of- plane bending	Branch Run	İ <sub>ob</sub> İ <sub>or</sub>	Note 1 0.605	1.40 -0.237	-0.558 0.528	0.406 1.42
Torsion	Branch	i <sub>tb</sub>	0.850	1.00	-0.50	2.10
	Run	i <sub>tr</sub>	0.864	-0.0473	0.543	0.609

Note 1: Replace  $A_{o}$  with 1.28[1.28(r/R)-(r/R)<sup>4</sup>] Note 2: T\* = T+ t<sub>r</sub>/2

#### 5.2 Combination Of Moments

For combination of stresses due to different moments, the various code committees are considering changes at this time. Until these changes are finalized, the evaluation of the stresses must be based on the maximum i-factor for the branch and the maximum i-factor of the run. If Section III is the Code of record (for Class 2 or 3 piping), Equation 3-2 should be used for evaluating the branch connections where the branch and each end of the run pipe are evaluated.

Conclusions

$$S = \frac{i(M_i^2 + M_o^2 + M_t^2)^{1/2}}{Z_i}$$
 (eq. 3-2)

The values of the moments and section modulus are appropriate for the location. The value of  $Z_i$  for the branch is given by equation :

$$z = \pi r^2 t \tag{eq. 3-3}$$

For the run pipe,  $Z_i$  is:

$$Z = \pi R^2 T \tag{eq. 3-4}$$

For the branch, the value of i is the maximum of  $i_{ib}$ ,  $i_{ob}$ , or  $i_{tb}$ . For the run, the value of i is the maximum of  $i_{ir}$ ,  $i_{or}$ , or  $i_{tr}$ .

If B31.1 is the Code of record, a less conservative approach is permitted. This is discussed in detail in reference [1].

#### 5.3 Flexibility Factors

The flexibility modeling of the branch connection should be based on Figure 4-1 for the branch. As indicated in the figure, it assumed that a point spring represents the connection. The flexibility factors of the point spring are given by:

	$\mathbf{k}_{xy} = \mathbf{B}_{o}$ (I	$D/T^{*})^{n1}($	d/D) <sup>n2</sup> $(d/t)$ <sup>n</sup>	3		(eq
	Part	k	B <sub>o</sub>	$\mathbf{n}_{1}$	$n_2$	n <sub>3</sub>
In-plane bending	Branch	$\mathbf{k}_{_{\mathrm{ib}}}$	0.488	1.279	0.391	-0.602
Out-of- plane bending	Branch	$\mathbf{k}_{_{\mathrm{ob}}}$	Note 1	1.72	0.5057	-0.717
Torsion	Branch	$\mathbf{k}_{_{tb}}$	2.43	0.751	2.11	-0.553

Note 1. Replace  $B_0$  with 0.828 (3.0 (d/D)-3.75(d/D)<sup>2</sup>+(d/D)<sup>3</sup>)

Note 2.  $T^* = T + t_r/2$ 

Note 3. It is assumed that pad-reinforced branch connection has no effect on the flexibility of the run pipe. The values of k for the point spring for the run pipe are set equal to zero.

#### 5.4 Applicability of Results

The equations specified above in 5.1 and 5.3 are applicable to the following range of parameters:

$3.75 \leq R/T \leq 49.5$	(eq. 3-5)
$3.75 \le r/t \le 99$	(eq. 3-6)
$0.125 \leq r/R \leq 1.0$	(eq. 3-7)

This is discussed in detail in reference [1].

In addition, the following limitations are specified for the pad configuration:

$0 \leq$	$t_r/T \le 1.25$	(eq. 3-11)
$\geq 0$	$D_{op}/d_o \leq 2.0$	(eq. 3-12)

#### 5.5 B Indices

This report is focused on SIFs and flexibility factors for ASME Section III Class 2 or 3 and ANSI B31.1 piping. Certain editions of ASME Section III require the use of B indices for the qualification of Class 2 or 3 piping. The scope of this report did not address the B indices.

If B indices are required in the qualification process, it is suggested that the existing plant-specific Code of record be followed.

#### 5.6 Comparison to Present Code

As discussed in reference [1], the equations developed for SIFs for unreinforced fabricated tees are more conservative than the present Code for certain geometry parameters. This is intended to cover certain unconservatism in the Code, which has been documented in various previous studies. This unconservatism has been addressed by the equations developed in reference [1] and used as the basis for the equations herein.

The use of the expressions defined in this report will result in a more realistic evaluation of pad-reinforced branch connections.

# **6** REFERENCES

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# **A** MATERIAL CERTIFICATIONS

Material Certifications

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Figure A-1 Certificate of Tests

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Figure A-2 Certificate of Tests Material Certifications

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# **B** CALIBRATION RECORD

**EPRI Licensed Material** 

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Calibration Record

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ExcRetion Gain AK Shuni Cal Cal Factor Phase Zaro Offsol Notes: Performed by:	10.883 <u>IVin</u> On In + 0.939 <i>IVin</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVa</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i> <i>TVA</i>		10,223 Né In 6,039 NA NA NA NA NA NA	20 Start D Ret 0 Bert 0 -20 -40 -90 -100		1.994 V 0.000 V 0.0		Date	5: 8/28/95 5: 0/28/90	1.994 V 0.000 V 0.0	-0.31 +0.1( +0.1) +0.1( +0.1) +0.1( +0.1)
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Calibration Report

**Calibration Record** 

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MT8 Systems Corporation 14000 Technology Drive Eden Prekle, MN 55344-2280

MTB

#### **Calibration Report**

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				-100	1	-10.012 V	+0.12%	1		1	1

BERNOW Performed by:

Dala; 0/28/96 Next Recommonded Calibration Due Date:

6/29/95

\* MTS Measurement Standards are Traceable to the National Institute of Standards and Technology. MT8 Force Standards are temperature compensated in the range spootfied by the manufacturer.

Perm Number 114093-11D (04-94)-PC

Figure B-2 **Calibration Report** 

# **C** TEST DATA AND RESULTS

#### **Overview of Appendix C**

The description of the testing is contained in Section 2. Sections 2.2.4 and 2.2.5 contain a summary of the results. This Appendix contains more detail regarding the test data. For each of the four tests (E, F, G, H), the following is provided:

1. Standard sheets containing load-displacement data

Data sheets are provided for the two loading directions: "Positive Loading" and "Negative Loading", (for example, up and down). Both the "Loading" and "Unloading" conditions for each of the directions are included for a total of four data sheets. The sheets are used to determine the linear slopes of the load-deflection curves for the four loading conditions.

The data includes loads, deflections, and so on. The columns identified as "modified" are for the case where adjustments are required to the data, such as resetting a dial gauge, and so on. No modifications were required for these tests.

2. Summary Load Deflection Sheet

This sheet contains a plot of the load defection curve and the four straight lines determined from the load-displacement data (item 1 above). This plot indicates the reasonableness of the slope of the load deflection curves.

3. Fatigue Test Data Analysis

This sheet contains the displacements and number of cycles at each displacement (if more than one displacement is used.) The calculations of the SIF are included. The stiffness used in the calculation is the average value of the four loading conditions (in item 1). Equivalent cycles are also calculated.

### Test Specimen E Test Report







NOTES:

C-5

**FATIGUE - LOAD DEFLECTION CURVE** 

TYPE:

REINFORCED BRANCH-E

TEST #:

		200	8	0.200 0.000	EOD			1000			1500			2000			2600		0000	0002							STION	
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NEGATIVE LO				00 -1.600 -1.400											\ 					•							+ TES'	
		L.,	I			L		(	sa	Nſ	104	4) C	JAG	רכ						<u> </u>			4					
	1.383	NOMINAL	(KSI)	-37.8	-36.5	-33.0	-30.9	-28.1	-25.7	-23.1	-20.2	-18.0	-15.4	-12.9	-10.2	-8.1	-5.1	-2.4	0.0									
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TA POINTS 1885	(IN) = 40		F (LBS)	-3002.4	-2855.9	-2526.3	-2321.3	-2057.6	-1837.9	-1603.5	-1347.1	-1171.4	-973.6	-790.5	-600	-461	-285	-124	Ž.									
BASED ON N DA : OF "m" =	۔ ،	MODIFIED	ہ (INCHES)	-1.675	-1.625	-1.49	-1.407	-1.297	-1.203	-1.103	-0.99	-0.906	-0.804	-0.707	-0.602	-0.519	-0.403	-0.298	-0.204									
'm" TO BE I THE VALUE	'SI, M=F x L	ED LOAD	F (LBS)	-3002.41	-2855.93	-2526.34	-2321.26	-2057.59	-1837.86	-1603.48	-1347.14	-1171.36	-973.601	-790.496	-600.066	-460.906	-285.125	-123.992	0.52									
- L	STRESS = M/Z K	MEASURE	δ (INCHES)	-1.68	-1.63	-1.49	-1.41	-1.30	-1.20	-1.10	-0.99	-0.91	-0.80	-0.71	-0.60	-0.52	-0.40	-0.30	-0.20									
= Fo + m	JOMINAL		 #	F	2	3	4	5	9	7	80	6	10	11	12	13	14	15	16	17	8	19	20	21	22	23	24	25

NOTES: Deflection is in an upward direction.

C-6

TEST #:	<b>REINFO</b>	RCED BRA	NCH-E			
C	OMPONENT:	REINFORCED BR	ANCH CONNE	CTION		
STIFFNE	ESS (lbs/in) =	<u>1963</u>	MC	DMENT ARM (in)=	<u>46.125</u>	
D (in) =	<u>4.5</u>	t (in) =	<u>0.237</u>		$Z(IN^3) = \pi r_n^2 t =$	<u>3.383</u>
TEST DI	SPLACEMEN	T/CYCLE DATA:				
	CONDITION	DISPLACEMENT	EFFECTIVE	NOMINAL	NUMBER	
	#	AMPLITUDE	APPLIED	STRESS	OF TEST	
		(+/-) (in.)	LOAD (lbs)	(+/-) (psi)	CYCLES	
		δί		S	Ni	
	1	1.00	1963	26,773	1,260	
	2	0.00	0	0	0	
	3	0.00	0	0	0	
	4	0.00	0	0	0	
	5	0.00	0	0	0	
	6	0.00	0	0	0	
	7	0.00	0	0	0	
	8	0.00	0	0	0	
				TOTAL CYCLES:	1,260	
THE EQ	UIVALENT NL	IMBER OF CYCLE	S, BASED ON	A DISPLACEME	δ <sub>max</sub> =	1
						INCHES
	IS: N <sub>eq</sub> = S	$SUM(\delta_i/\delta_{max})^5 * N_i =$	1,260			
			i = 245	5,000 * N <sub>eq</sub> <sup>(-0.2)</sup> /S =	2.195	
FOR NC	MINAL DIMEN	NSIONS: $Z(IN^3) =$	<u>3.215</u>	i =	2.309	
COMME	NTS:					
1. HT #	0947853 (Bra	nch), #D947853B (	Run)	-	-	
2. L is c	listance to wel	d on branch. Dista	ance to center	of weld on branch	is 46-1/8 in.	
Distanc	e to center of	weld on large pipe v	vas 46-7/8 in.			

### Test Specimen F Test Report



**FATIGUE - LOAD DEFLECTION CURVE** 

түре:

REINFORCED BRANCH-F

TEST #:



NOTES:

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NOTES:

**FATIGUE - LOAD DEFLECTION CURVE** 

TYPE:

**REINFORCED BRANCH-F** 

TEST #:



Deflection is in an upward direction.

C-12

EATIC			e			
FAIIG		ATA ANAL I SI	3			
TEST #:	<u>REINFO</u>	RCED BRA	<u>NCH-F</u>			
C	OMPONENT:	REINFORCED BR	ANCH CONNE	<u>ECTION</u>		
STIFFNE	ESS (lbs/in) =	<u>2018</u>		MOMENT ARM (in)=	<u>46.25</u>	
D (in) =	<u>4.5</u>	t (in) =	0.237		$Z(IN3) = \pi r_{n2}t =$	<u>3.383</u>
TEST DI	SPLACEMEN	T/CYCLE DATA:				
	CONDITION	DISPLACEMENT	EFFECTIVE	NOMINAL	NUMBER	
	#	AMPLITUDE	APPLIED	STRESS	OF TEST	
		(+/-) (in.)	LOAD (lbs)	(+/-) (psi)	CYCLES	
		δι		S	Ni	
	1	0.80	1614	22,067	2,216	
	2	0.87	1755	23,997	444	
	3	0.00	0	0	0	
	4	0.00	0	0	0	
	5	0.00	0	0	0	
	6	0.00	0	0	0	
	7	0.00	0	0	0	
	8	0.00	0	0	0	
				TOTAL CYCLES:	2,660	
THE EQ	UIVALENT NU	JMBER OF CYCLE	S, BASED ON	A DISPLACEMENT	δ <sub>max</sub> =	0.87
						INCHES
	IS: N <sub>eq</sub> = S	$SUM(\delta_i/\delta_{max})^5 * N_i =$	1,901			
			i =	245,000 * N <sub>eq</sub> <sup>(-0.2)</sup> /S =	2.255	
FOR NC	MINAL DIMEN	NSIONS: $Z(IN^3) =$	<u>3.215</u>	i =	2.373	
COMME	NTS:					
1. HT #	<sup>±</sup> 0947853 (Bra	anch), #D947853B (	Run)			
2. At 0	deflection the	load is 110 pounds.				
3. Crac	ks were visible	at the toe of the w	eld on the brar	nch at the bottom.		
	1				I	

### Test Specimen G Test Report





NOTES:



NOTES:

FATIGUE - LOAD DEFLECTION CURVE

TYPE:

**REINFORCED BRANCH-G** 

TEST #:



NOTES: Deflection is in an upward direction.

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C-18

FATIG	UE TEST D	ATA ANALYSI	S			
TEST #:	<u>REINFO</u>	<u>RCED BRA</u>	<u>NCH-G</u>			
C	OMPONENT:	REINFORCED BR	ANCH CONNE	ECTION		
STIFFNE	SS (lbs/in) =	<u>2109</u>		MOMENT ARM (in)=	<u>46.125</u>	
D (in) =	<u>4.5</u>	t (in) =	0.237		$Z(IN3) = \pi r_n^2 t =$	<u>3.383</u>
TEST DI	SPLACEMEN	I/CYCLE DATA:				
	CONDITION	DISPLACEMENT	EFFECTIVE	NOMINAL	NUMBER	
	#	AMPLITUDE	APPLIED	STRESS	OF TEST	
		(+/-) (in.)	LOAD (lbs)	(+/-) (psi)	CYCLES	
		δί		S	Ni	
	1	1.00	2109	28,752	1,366	
	2	0.00	0	0	0	
	3	0.00	0	0	0	
	4	0.00	0	0	0	
	5	0.00	0	0	0	
	6	0.00	0	0	0	
	7	0.00	0	0	0	
	8	0.00	0	0	0	
				TOTAL CYCLES:	1,366	
						4
THE EQ	UIVALENT NU		S, BASED OF		δ <sub>max</sub> =	1
		N IN 4/0 /0 \5 * N	4 000			INCHES
	IS: N <sub>eq</sub> = 5	$SUM(\delta_i/\delta_{max})^{\circ} N_i =$	1,366			
				0.45 000 * NL (-0.2)/C	0.014	
			1=	245,000 " N <sub>eq</sub> " /S =	2.011	
FOR NO		$SIONS: Z(IN^3) =$	3.215	i =	2.116	
COMME	NTS:					
1. HT #	0947853 (Bra	nch), #D547853B (l	Run)			
2. L is c	listance to wel	d on branch. Dista	ance to center	of weld on branch is	46-1/8 in. Dis	tance
to cente	er of weld on la	rge pipe was 47-5/8	3 in.			
3. Failu	re occured at t	he top and bottom	at the same ti	me at the toe of the w	eld on the bra	nch
1 · · · ·		I	1	1	1	

### Test Specimen H Test Report



**FATIGUE - LOAD DEFLECTION CURVE** 

ТҮРЕ:



NOTES:



NOTES:

**FATIGUE - LOAD DEFLECTION CURVE** 

ТҮРЕ:

**REINFORCED BRANCH-H** 

TEST #:

		ŗ		+	0.00		Ļ					4			1			1			Ļ			1			r		-7
NOL					0 -0.400 -0.200				100			-150			-200									A09				LINEAR-DEFLECTION	
LIQNO					9.0			Ì	7																ES)	•		ł	
ing c		-		+	-0.800				Y	4	<u> </u>	-			_										N (INCH			NO	
LOAD		-		_	-1.000						/	Y	7		_			_							LECTIO			FLECT	
ND - C		-			50				_			_	4												DEF			OAD-DE	
LOAI					5											٩												TESTL	
LTIVE		-			7														<u> </u>									ł	
NEG/		-		-	-1.60													+	7			~					-		
		L			-1.800			•						-						_				]					
									(	sa	Nſ	104	4) C	JAG	51														
01	3.383	NOMINAL	STRESS	(KSI)	-39.3	-36.3	-33.5	-30.9	-28.6	-25.8	-23.3	-20.6	-17.9	-15.4	-13.2	-10.0	-7.6	-5.0	-2.2	0.0									
Fo (LBS) =	iN3) =ѫí <sup>,2</sup> t=	L.	BASED	(LBS)	-2925	-2703	-2494	-2299	-2125	-1918	-1731	-1532	-1333	-1141	626-	-743	-562	-371	-166	0									
<b>4</b> )	7	OR		E E	2093	2032	1991	1951	1913	1872	1836	1806	1785	1773	1764	1760	1745	1732	1810	-	-								
uTS, N =	45,5	SLOPE F	START DATA DC	(LBS/IN																N/A									
ATA POII 1951	= (NI)-		LOAD	(LBS)	-3229.9	-2849	-2578	-2329	-2116.6	-1867.6	-1647.9	-1428.1	-1223.1	-1039.9	-886.14	-674	-505	-329	-154	0									
Based on N D/ E of "m" =	۔ ت	MODIFIE		(INCHES)	-1.721	-1.607	-1.5	-1.4	-1.311	-1.205	-1.109	-1.007	-0.905	-0.807	-0.724	-0.603	-0.51	-0.412	-0.307	-0.222									
m" TO BE HE VALUI	(SI, M=F x	0	LOAD	(LBS)	3229.89	-2849.03	-2578.03	-2329.01	-2116.61	-1867.58	-1647.86	-1428.13	-1223.05	-1039.95	-886.136	-673.734	-505.277	-329.495	-153.714	0.094									
۶u	STRESS = M/Z }	MEASURE		(INCHES)	-1.721	-1.607	-1.5	-1.4	-1.311	-1.205	-1.109	-1.007	-0.905	-0.807	-0.724	-0.603	-0.51	-0.412	-0.307	-0.222									
= Fo + r	JOMINAL	DATA	OINT	#	-	2	9	4	5	9	2	8	6	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25

NOTES: Deflection is in an upward direction.

FATIG	UE TEST D	ATA ANALYSI				
TEST #:	REINFO	<b>RCED BRA</b>	NCH-H			
C	OMPONENT:	REINFORCED BR	ANCH CONNE	CTION		
STIFFNE	ESS (lbs/in) =	<u>2092</u>		MOMENT ARM (in)=	<u>45.5</u>	
D (in) =	<u>4.5</u>	t (in) =	0.237		$Z(IN3) = \pi r_n^2 t =$	<u>3.383</u>
TEAT D						
IEST DI	SPLACEMEN	I/CYCLE DATA:				
				ΝΟΜΙΝΑΙ		
				STRESS		
	π	(+/-) (in )		(+/-) (nsi)	CYCLES	
		δ.		S	N:	
	1	1 00	2092	28 133	1 075	
	2	0.00	0	0	0	
	3	0.00	0	0	0	
	4	0.00	0	0	0	
	5	0.00	0	0	0	
	6	0.00	0	0	0	
	7	0.00	0	0	0	
	8	0.00	0	0	0	
				TOTAL CYCLES:	1,075	
IHE EQ	UIVALENT NU	IMBER OF CYCLE	S, BASED ON	A DISPLACEMENT	δ <sub>max</sub> =	1
			4.075			INCHES
	IS: N <sub>eq</sub> = 5	$\delta UM(\delta_i/\delta_{max})^{\circ} \cap N_i =$	1,075			
				0.45 000 * NL (-0.2)/O	0.450	
			1 = .	245,000 " N <sub>eq</sub> '/S =	2.156	
		1910NIQ: 7/INI <sup>3</sup> ) -	2 215	i	2 260	
FUR NC		νοισίνο. ζ(IIN ) =	3.213		2.209	
COMME	NTS:					
1 HT +	10047853 (Bra	   #D5/7853B (	Run)			
2   is c	listance to we	Id on branch Dist	ance to center	of weld on branch is	46 in Distan	ce to
center	of weld on large	e pipe was 46-5/8 ir	1.			